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## Thermodynamic performance of IGCC with oxy-combustion CO<sub>2</sub> capture

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## Purpose of the study

- CO<sub>2</sub> capture from coal power plant. Configurations proposed:
  - USC with amine capture or with oxy-combustion
  - IGCC with pre-combustion capture
- Investigate novel plant configurations, to obtain better thermodynamic and environmental performance
- Near-term solutions requiring the development of components not available in today's marketplace, but not requiring unproven technologies
- Investigated configurations:
  - Reference IGCC, without and with capture
  - Oxy-combustion, with semi-closed H<sub>2</sub>O-CO<sub>2</sub> power cycle, present technology
  - Oxy-combustion, with co-capture CO<sub>2</sub>-SO<sub>2</sub>, advanced technology
  - Oxy-combustion, Hot Gas Desulfurization, advanced technology
- Detailed thermodynamic analysis and optimization



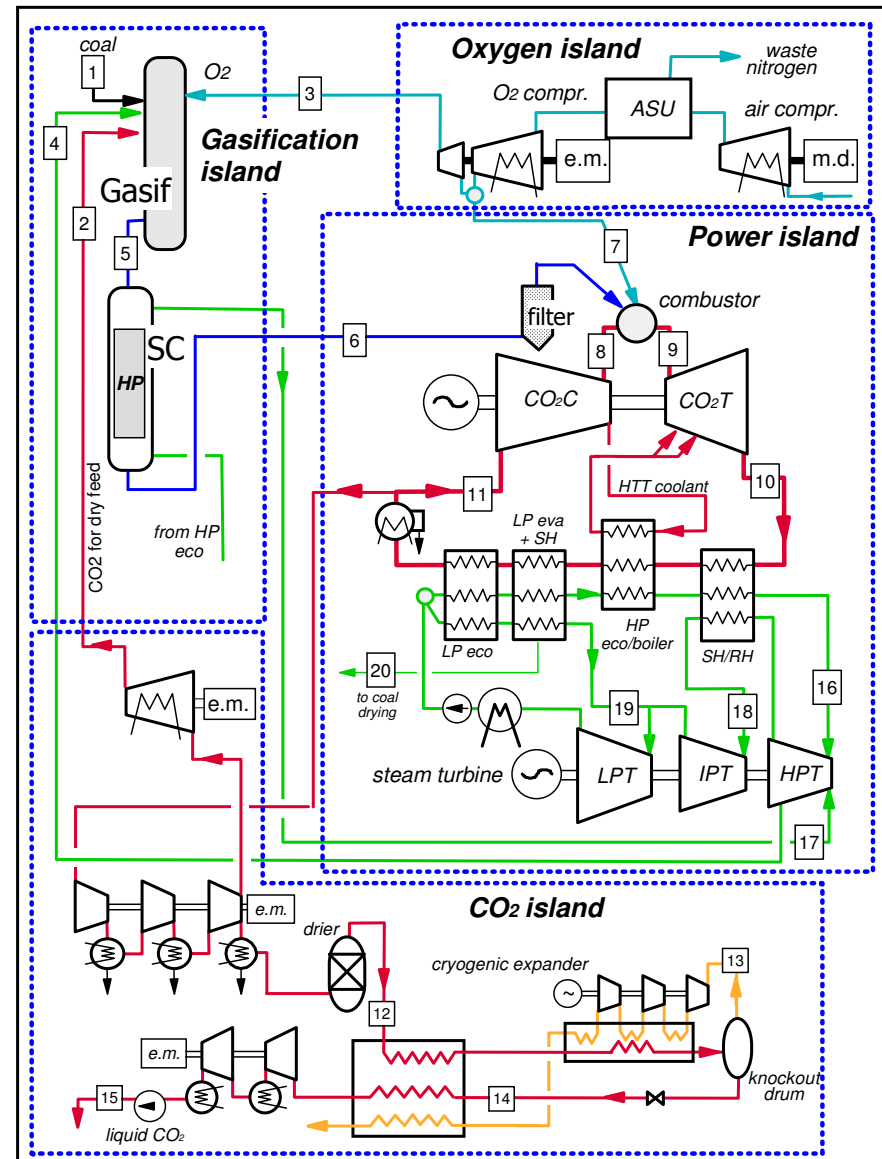
## The reference cases

- Calculations by means GS code, developed in our Department and used for power plant on-design performance prediction
- No capture: A status-of-the-art IGCC combined cycle based:
  - An oxygen-blown dry-feed entrained-flow gasifier (Shell type)
  - Syngas cooling by syngas quench and HP steam gen.
  - Power cycle based on a FB-technology gas turbine
- With pre-combustion capture:
  - Same basic plant configuration and calculation assumptions
  - CO conversion to hydrogen by means of two Water-Gas-Shift reactors (HT: bulk conversion, LT: 'finishing')
  - H<sub>2</sub>S/CO<sub>2</sub> removal by Selexol in two separate absorbers; largest solvent fraction for CO<sub>2</sub> removal with pressure-swing regeneration; smaller fraction for H<sub>2</sub>S removal with stripper regeneration.



# Oxy-fuel IGCC

- Syngas from conventional gasification is burnt with high-purity oxygen in a semi-closed CO<sub>2</sub>-H<sub>2</sub>O Joule cycle:
  - CO<sub>2</sub> to lock-hoppers.
  - A CO<sub>2</sub> compressor recycles CO<sub>2</sub> to the oxy-combustor
  - Larger pressure ratio to optimize cycle efficiency
- A CO<sub>2</sub> cryogenic purification system is needed to eliminate incondensable gases during compression

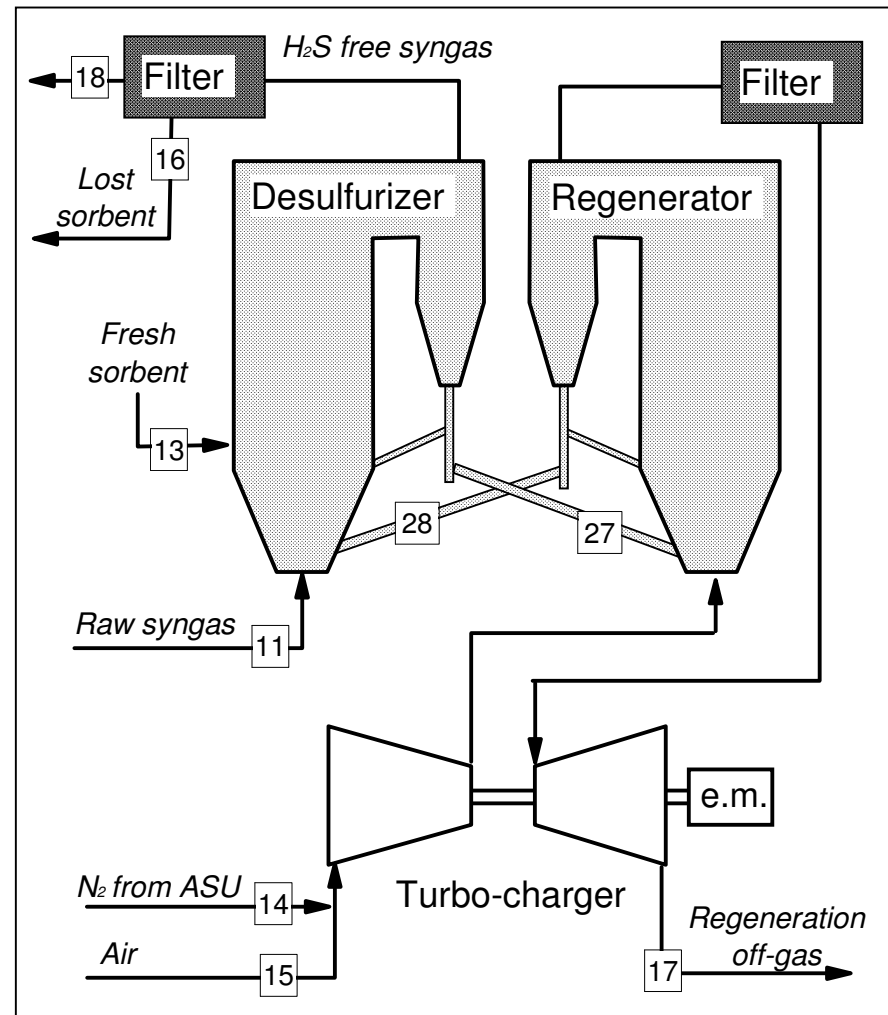




- The gas turbine must be re-designed, using known methodologies and present technology. Large development costs can be anticipated.
- Present technology:
  - Conventional gasification with H<sub>2</sub>S separation by Selexol
- Advanced technology with CO<sub>2</sub>-SO<sub>2</sub> co-sequestration:
  - High pressure gasifier
  - Hot Gas Filtration (550°C)
  - H<sub>2</sub>S not separated (sent to burner)
  - Improved gas turbine, due to the mid-long term application
  - Blade coolant cooling to improve TIT



- To avoid Co-sequestration of CO<sub>2</sub> and SO<sub>2</sub>
- Reactions
  - $\text{ZnO} + \text{H}_2\text{S} \rightarrow \text{ZnS} + \text{H}_2\text{O}$
  - $\text{ZnS} + 3/2 \text{O}_2 \rightarrow \text{ZnO} + \text{SO}_2$
- Desulfurization @550°C
- Regeneration @750°C
- Regeneration gas with 2% O<sub>2</sub> to avoid ZnSO<sub>4</sub> formation





- Performance calculated by GS code, developed at Energy Dept, Politecnico di Milano
- Used since two decades to calculate the performance of any type of power plant
- Built-in correlations for components' efficiency prediction
- GT blade cooling models
- Assumptions from literature and industrial experience

<i>Gasification and ASU</i>		<i>Gas turbine and steam cycle</i>	
Gasification pressure, bar	44	Fuel temperature, °C	250
Gasification temperature, °C	1550	GT turbine inlet temperature, °C	1335
Heat losses, % LHV	0.7	GT pressure ratio	17
Carbon conversion	99.0	Pressure levels, bar	130/36
Temperature of O <sub>2</sub> to gasifier, °C	15	SH/RH temperature, °C	565
Moderator steam, kg <sub>H<sub>2</sub>O</sub> /kg <sub>coal</sub>	0.06	Pinch point/sub-cooling ΔT, °C	10/5
N <sub>2</sub> to lock hoppers, kg/kg <sub>dry-coal</sub>	0.22	Condensing pressure, bar	0.04
Quenched syngas temperature, °C	900	Minimum stack temperature, °C	115
Cold recycle syngas temperature, °C	200	<i>CO<sub>2</sub> compression</i>	
Min. ΔT in syngas coolers, °C	20	Number of inter-cooled stages	5
Oxygen purity, % mol.	95	Inter-cooling temperature, °C	25
ASU electric consumption, kWh/t <sub>O<sub>2</sub></sub>	325	Inter-coolers pressure loss, %	1
		Compressors isentropic efficiency, %	82

Table 1 – Assumptions for the reference IGCC plants, present technology.

<i>Water Gas Shift Reactors</i>		<i>Selexol Plant</i>	
Steam to carbon at first reactor inlet	1.5	L/G ratio (wt. basis) in H <sub>2</sub> S/CO <sub>2</sub> absorption columns	1.1/11.6
HT reactor outlet temperature, °C	400	CO <sub>2</sub> flash tanks pressures, bar	15/8/3.5/1.5
LT reactor outlet temperature, °C	210	Reboiler heat duty, MW <sub>th</sub>	27

Table 2 – Additional assumptions for the IGCC plant with pre-combustion capture.

CO <sub>2</sub> to gasifier lock hoppers, kg/kg <sub>coal</sub>	0.35	Fuel side pressure loss at combustor, %	20
GT pressure ratio	40	O <sub>2</sub> content at combustor outlet, % mol.	2

Table 3 – Varied assumptions for the oxy-fuel IGCC plant, present technology.

Gasification temperature, °C	1427	Syngas temperature to GT, °C	550
Gasification pressure, bar	50	Steam pressures HP/RH, bar	247/58
Carbon conversion	99.5	LP evaporation pressure, bar	4
Temperature of O <sub>2</sub> to gasifier, °C	200	SH/RH steam temperature, °C	600/600

Table 4 – Varied assumptions for the oxy-fuel IGCC plant, advanced technology.

ZnO to TiO <sub>2</sub> mol.ratio in fresh sorbent	1	O <sub>2</sub> mol.fraction in regeneration mixture	2%
System pressure, MPa	≈ 5	Regeneration temperature, °C	750
Desulphurization temperature, °C	550	ZnS to ZnO mol.ratio in regen.sorbent	0.1
Sorbent loss, % in wt.	0.1	Pressure loss at the hot gas filter, %	3

Table 5 – Assumptions for Hot Gas Desulfurization.



# Results of the performance analysis

<i>Case</i>	<i>Refer- ence IGCC</i>	<i>Pre- combus- tion IGCC</i>	<i>Oxy- IGCC</i>	<i>Advanced oxy-IGCC</i>	<i>Advanced oxy-IGCC + HGD</i>
TIT, °C	1335	1335	1335	1400	1400
Sulfur co-sequestration	-	no	no	yes	no
Electric/mechanical power MW					
Gas turbine (2 units)	659.4	597.7	609.6	656.5	653.2
GT auxiliaries	-2.34	-2.12	-2.16	-2.30	-2.32
Steam Turbine	420.6	354.0	454.7	510.8	513.6
Steam cycle pumps	-6.53	-6.92	-6.51	-11.60	-11.01
ASU	-72.15	-72.17	-178.17	-188.2	-187.0
Lock hoppers N <sub>2</sub> compress.	-9.11	-9.11	-	-	-
Syngas recycle fan	-2.4	-2.41	-2.42	-	-
Syngas compressor	-	-	-10.79	-	-
N <sub>2</sub> compressor for fuel dilution	-69.13	-51.34	-	-	-
Aux. for H <sub>2</sub> S / CO <sub>2</sub> removal	-0.69	-26.16	-0.69	-	-3.40
CO <sub>2</sub> compression	-	-40.80	-83.05	-83.30	-83.30
Auxiliaries for heat rejection	-5.86	-5.18	-8.88	-11.03	-11.03
Miscellaneous BOP	-7.40	-7.38	-7.40	-7.47	-7.45
<b>Net power output, MW<sub>el</sub></b>	<b>904.4</b>	<b>728.2</b>	<b>764.2</b>	<b>863.5</b>	<b>861.3</b>
Fuel input LHV, MW <sub>th</sub>	1897.6	1897.6	1897.6	1897.6	1897.6
Cold Gas Efficiency, %	78.14	69.63	77.36	83.85	83.31
<b>Net LHV efficiency, %</b>	<b>47.66</b>	<b>38.38</b>	<b>40.27</b>	<b>45.50</b>	<b>45.39</b>
CO <sub>2</sub> captured, %	0	90.76	97.38	97.38	97.38
<b>CO<sub>2</sub> spec.emissions, g/kWh</b>	<b>732.1</b>	<b>82.3</b>	<b>25.1</b>	<b>22.2</b>	<b>22.3</b>

Table 6 –Performance of the plants considered in the paper.



## Conclusions

- Very interesting performance can be predicted for Oxy-fuel IGCC:
  - Environmental: NO<sub>x</sub> and SO<sub>x</sub> not wasted to the atmosphere, but mostly solved within the stream to sequestration
  - Environmental: 95-99% carbon capture, depending on the solutions adopted for incondensable gases stream
  - Efficiency: better than 45% by adopting advanced solutions, specific to oxy-fuel configurations (not applicable to decarbonization):
    - Hot Gas Filtration
    - Co-sequestration of CO<sub>2</sub>-SO<sub>2</sub> or Hot Gas Desulfurization
- Drawbacks: the gas turbine must be re-designed, using known methodologies and present technology.
- A larger attention may be devoted to Oxy-fuel IGCC in the R&D programs



# Oxy-fuel CO<sub>2</sub> capture: present technology

